

Hooke's Law: Nonlinear Generalization and Applications

Part II Parameters of the Bending Pressure Zone of a Beam

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Keywords: Center of stress β_0 , stress α_0 and strain ε_0 of the bending pressure zone in dependence on strength W and loading κ of concrete.

Introduction

The bending pressure zone of a pressed beam is limited by the pressed side and the neutral axis, where the strain ε is zero. If one takes, as is shown in figure 1, a prism, limited by these two boundaries, out of the beam and brings an eccentric loading to the ends, so that the strain ε on the section-side is zero, then one has in the prisma a copy of the distribution of the loading in the bending pressure zone ([3], p.1). See figure 1.

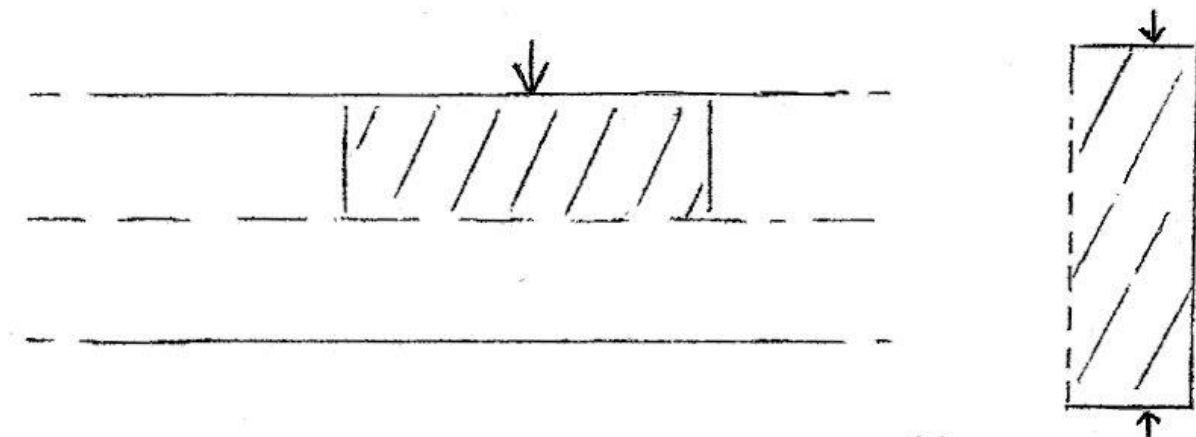


Fig. 1: Equivalence of the bending pressure zone of a beam and an eccentric pressed prism

In figure 2 a sketch of such a prisma is given (copy of [3]). For comparability I also use the symbols and measures of [3].

The problem of the extensive experiments in [3] was to find for different values of cube-strength $W(\text{kg}/\text{cm}^2$; side length 20 cm) and $\kappa = \bar{\sigma}/\bar{\sigma}_B$ (B for break/Bruch) the values of β_0 , the measure of eccentricity of the loading of the prism in [3] and center of the distribution of the stress, and of the (relative) stress $\alpha_0^P = \bar{\sigma}/K_b$, where K_b is the strength of the prism. These values β_0 , α_0^P and $\varepsilon_0 (= \varepsilon_1)$ are found, when, by varying W and/or κ , ε_2 becomes zero.

To this at first for a fixed value of W and a fixed value of β a series experiments was done with growing stress until break.

Then the same was done with prisms (of the same W) and other values of β .

At last step 1 and 2 were done with other values of W .

In detail see [3], figures 48-88.

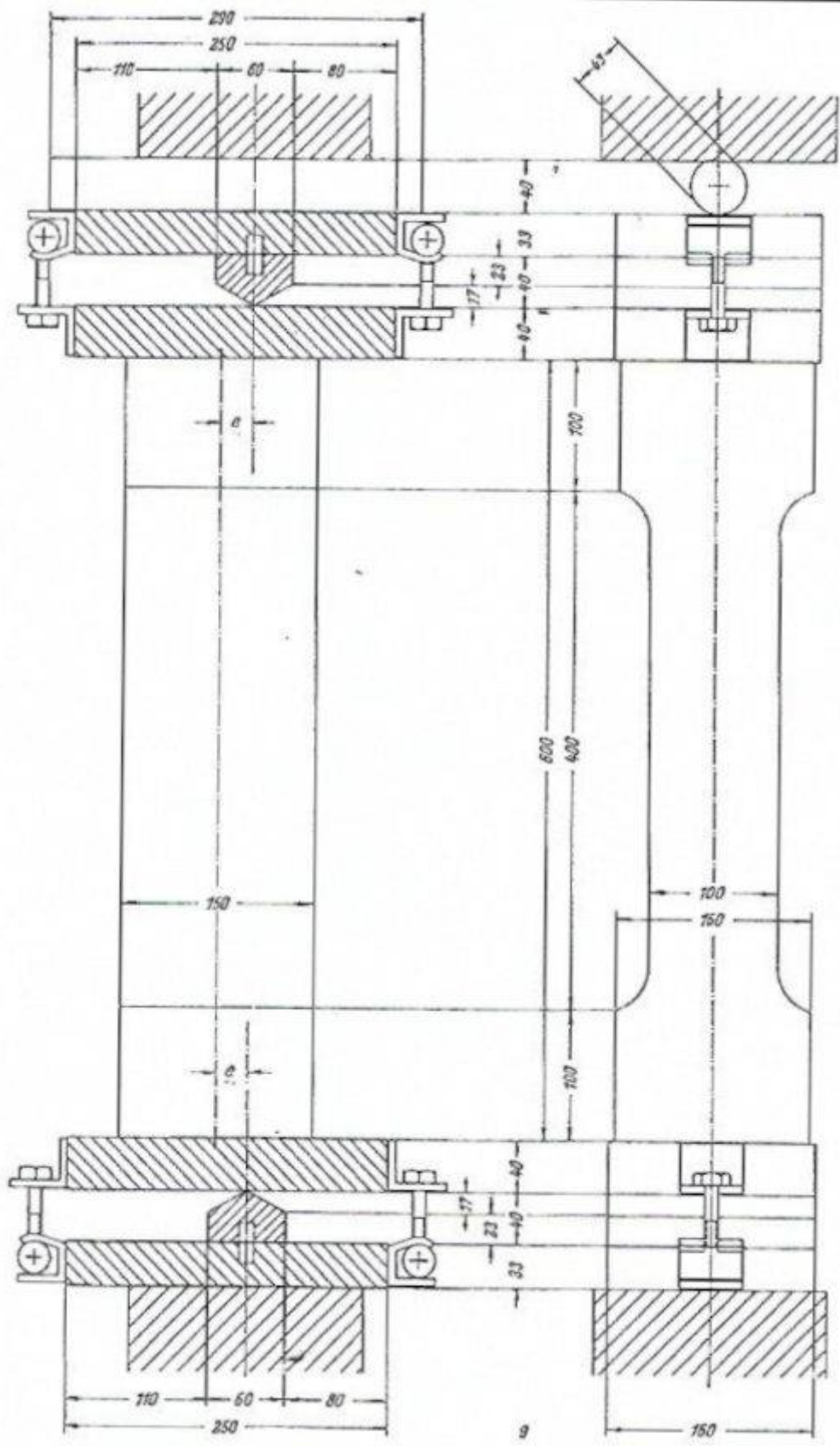


Fig. 2: Details of loading the prisms

With these data the results $\alpha^P(\beta)$, $\varepsilon_2(\beta)$ and $\varepsilon_1(\beta)$ for different values of κ were plotted for different values of W (see [3], figures 94-107). The essential results are the values $\beta=\beta_0$, where $\varepsilon_2=0$. Then $\alpha^P=\alpha_0^P$ and $\varepsilon_1=\varepsilon_0$. See figure 3.

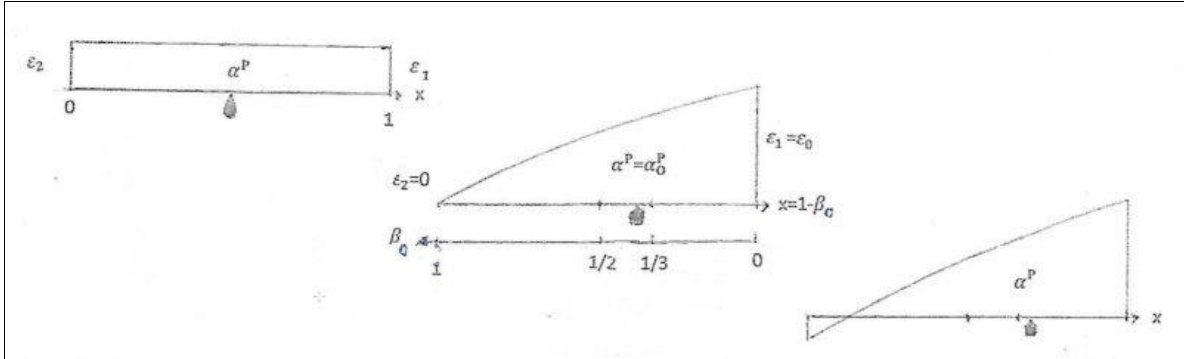


Fig. 3: Parameters strain ε , (rel) stress $\alpha_{p=\bar{\sigma}} / \kappa_b$ (κ_b is strength of prism) and measure of eccentricity β of loading. Eccentricity $e=d(0.5-\beta)$; $d=150\text{mm}$

These results are given in [3], Bild 113, 112, 114, repeated here as figures 4a, 5a, and 6a and listed in tables 1, 2, and 3. They are the experimental basis for the following analytical hypotheses.

To get trustworthy results it is of decisive importance to disregard non-truworthy experimental data. Of this sort are the experiments with high loading $\kappa > 0.9$. See e.g. [3]. p.24, or [2], p.74.

For this reason experimental data with $\kappa > 0.9$ were ignored in the tables and in the computations.

Hypotheses

Hypothesis H1 for β_0 (and the center of loading $(1-\beta_0)$)

$$\hat{\beta}_0(w, \kappa) = 1/3 + (1/2 - 1/3)e^{-a(W/100)} \kappa^b$$

The parameters a and b are computed with Gauss' method of least squares:

$Q(a, b) = \sum (\beta_{0i} - \hat{\beta}_{0i})^2 \rightarrow \text{Min. } (i=1, \dots, 49)$; the minimum is found with the iterative nonlinear simplex-method of Nelder and Mead [1]. We get:

$$\hat{\beta}_0 = 1/3 + 1/6 e^{-0.3374(W/100)} \kappa^{3.757}$$

The results are given in table 1, lines 2 and especially in figure 4b: $\hat{\beta}_0(W)$ for fixed values of κ and in figure 4c: $\hat{\beta}_0(\kappa)$ for fixed values of W . Notice the good correspondence of the hypothesis (curves) with the high number ($n=49$) of experimental points! Dotted curves are extrapolated.

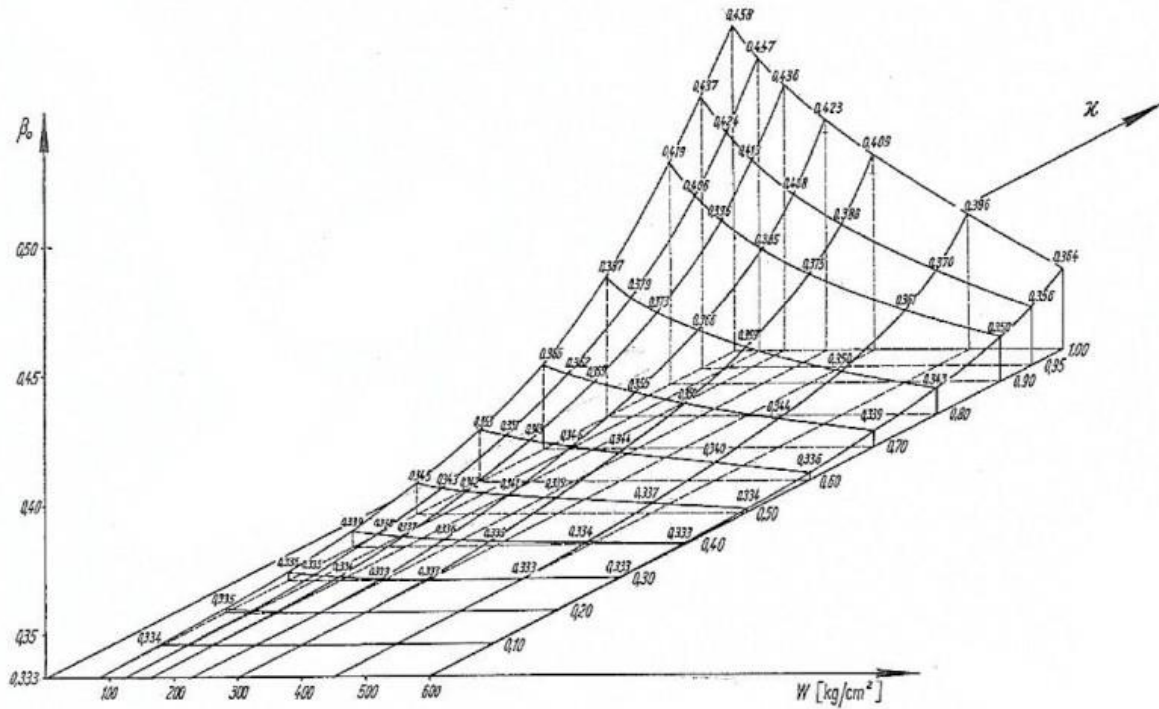


Fig. 4a: Experimental results $\beta_0(W, \kappa)$ according to [3]

Table 1: first lines: Experimental values β_0 ; second lines: hypothetical values of $\hat{\beta}_0$

W/100	.80	1.20	1.60	2.25.	3.00	4.50.	6.00
kappa							
0.3	0.336	0.335	0.334	0.333	0.333	0.333	0.333
	0.335	0.335	0.334	0.334	0.334	0.334	0.333
0.4	0.339	0.338	0.337	0.336	0.335	0.334	0.333
	0.337	0.337	0.336	0.336	0.335	0.334	0.334
0.5	0.345	0.343	0.342	0.341	0.339	0.337	0.334
	0.343	0.342	0.341	0.339	0.338	0.336	0.335
0.6	0.353	0.351	0.349	0.346	0.344	0.340	0.336
	0.352	0.350	0.348	0.344	0.342	0.339	0.337
0.7	0.365	0.362	0.359	0.355	0.350	0.344	0.339
	0.367	0.362	0.359	0.354	0.349	0.343	0.339
0.8	0.387	0.379	0.373	0.366	0.359	0.350	0.343
	0.388	0.381	0.375	0.367	0.360	0.349	0.343
0.9	0.419	0.406	0.396	0.385	0.375	0.361	0.360
	0.419	0.408	0.399	0.386	0.374	0.358	0.348

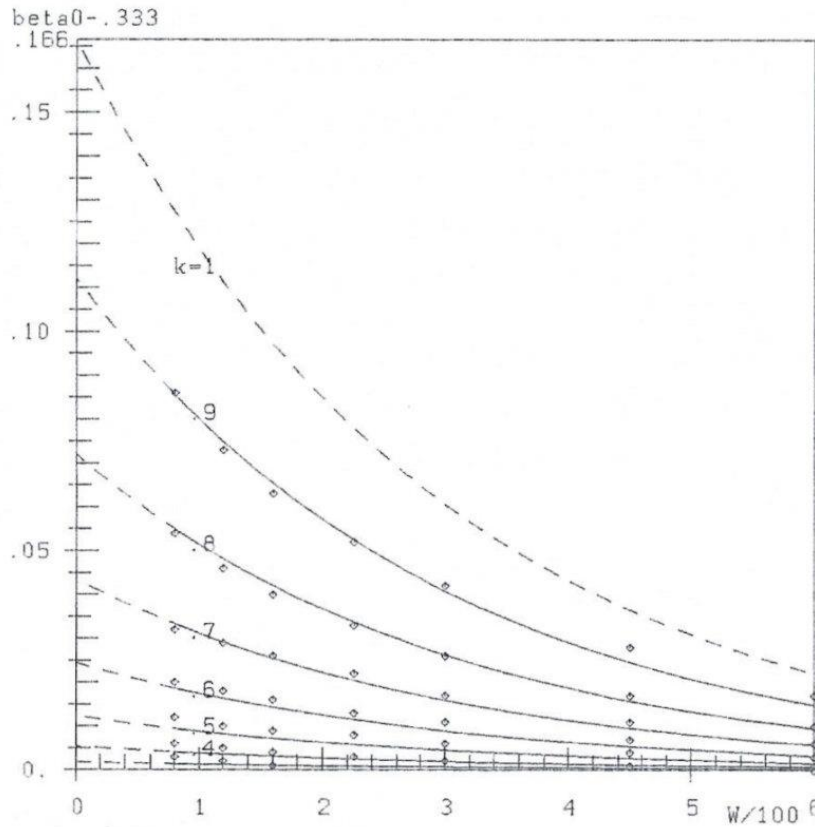


Fig. 4b: Experimental points ($W/100, \beta_0=0.333$) and hypothetical curves ($W/100, \hat{\beta}_0=0.333$)

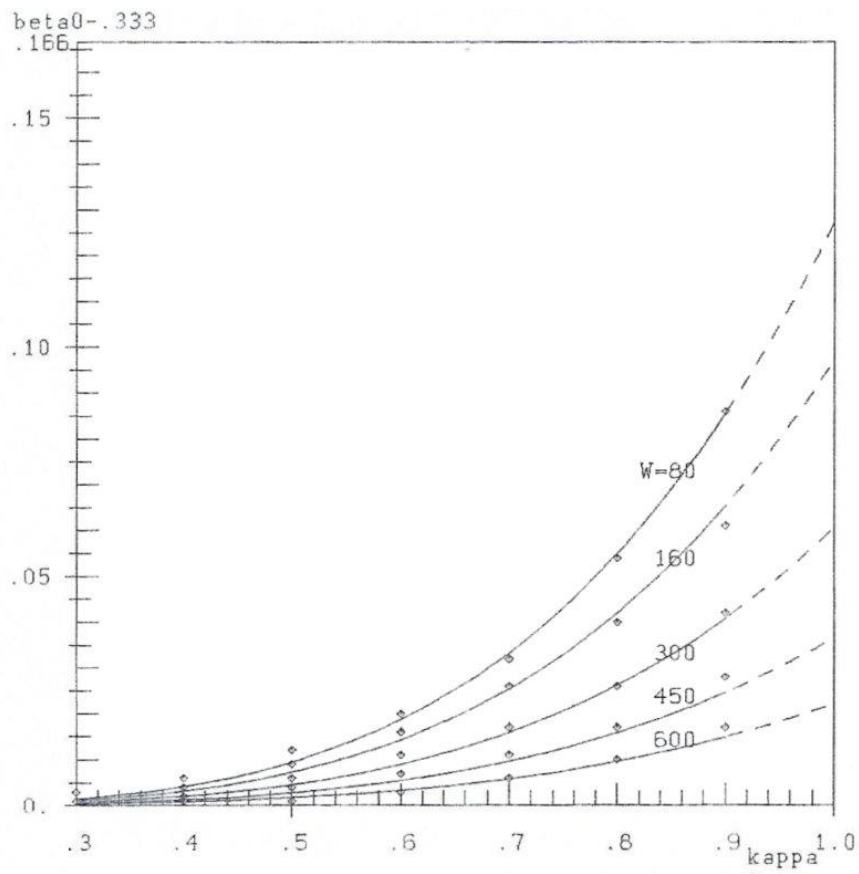


Fig. 4c: Experimental points ($\kappa, \beta_0=0.333$) and hypothetical curves ($\kappa, \hat{\beta}_0=0.333$)

Hypothesis H2 for the (relative) loading $\alpha_0^p = \bar{\sigma}_0 / K_b$:

$$\hat{\alpha}_0^p(W, \kappa) = (2/3)\kappa + (1/3)e^{-a(W/100)} \kappa^b$$

The analogous calculation as with hypothesis H1, now with the 49 data of table 2 gives:

$$\hat{\alpha}_0^p = (2/3)\kappa + 1/3 e^{-0.4052(W/100)} \kappa^{5.846}$$

The results are given in table 2, lines 2 and in figures 5b and 5c. Especially in figure 5b correspondence of data-points and hypothetical curves is very convincing– with one exception!

For high loading ($\kappa = 0.9$) and small values of W. For this also see [3], p.24.

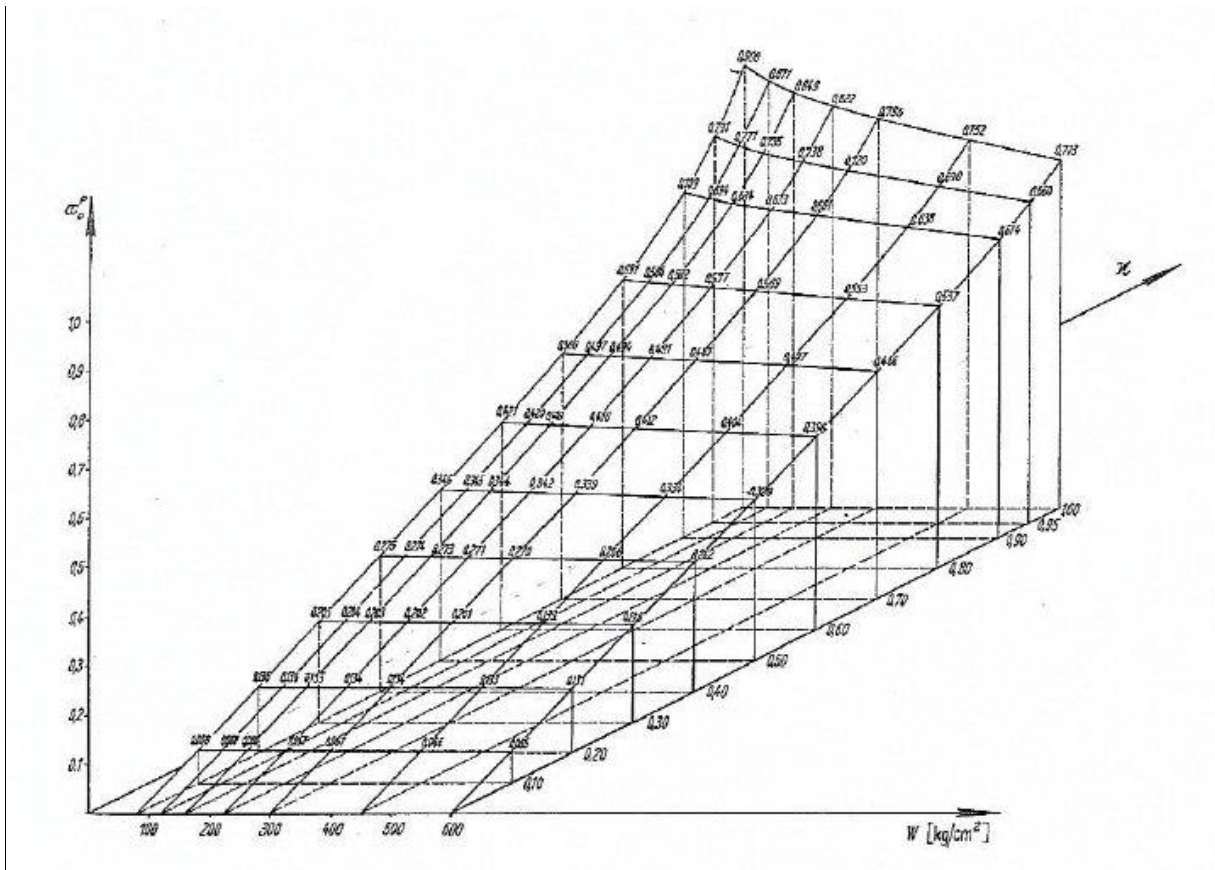


Fig. 5a: Experimental results $\alpha_0^p (W, \kappa)$ according to [3]

Table 2: First lines: experimental values α_0^P , second lines: hypothetical values $\hat{\alpha}_0^P$

W/100	0.80	1.20	1.60	2.25	3.00	4.50	6.00
kappa							
0.3	0.205	0.204	0.203	0.202	0.201	0.199	0.196
	0.202	0.202	0.202	0.201	0.201	0.200	0.200
0.4	0.275	0.274	0.273	0.271	0.270	0.266	0.262
	0.268	0.268	0.267	0.267	0.267	0.266	0.266
0.5	0.346	0.345	0.344	0.342	0.339	0.334	0.328
	0.338	0.337	0.336	0.336	0.335	0.334	0.333
0.6	0.421	0.420	0.418	0.416	0.412	0.404	0.396
	0.412	0.410	0.409	0.407	0.405	0.403	0.401
0.7	0.500	0.497	0.494	0.491	0.487	0.477	0.466
	0.497	0.492	0.488	0.483	0.479	0.473	0.470
0.8	0.591	0.586	0.582	0.577	0.569	0.553	0.537
	0.599	0.589	0.580	0.569	0.560	0.547	0.541
0.9	0.709	0.694	0.684	0.673	0.661	0.638	0.614
	0.730	0.710	0.694	0.672	0.653	0.629	0.616

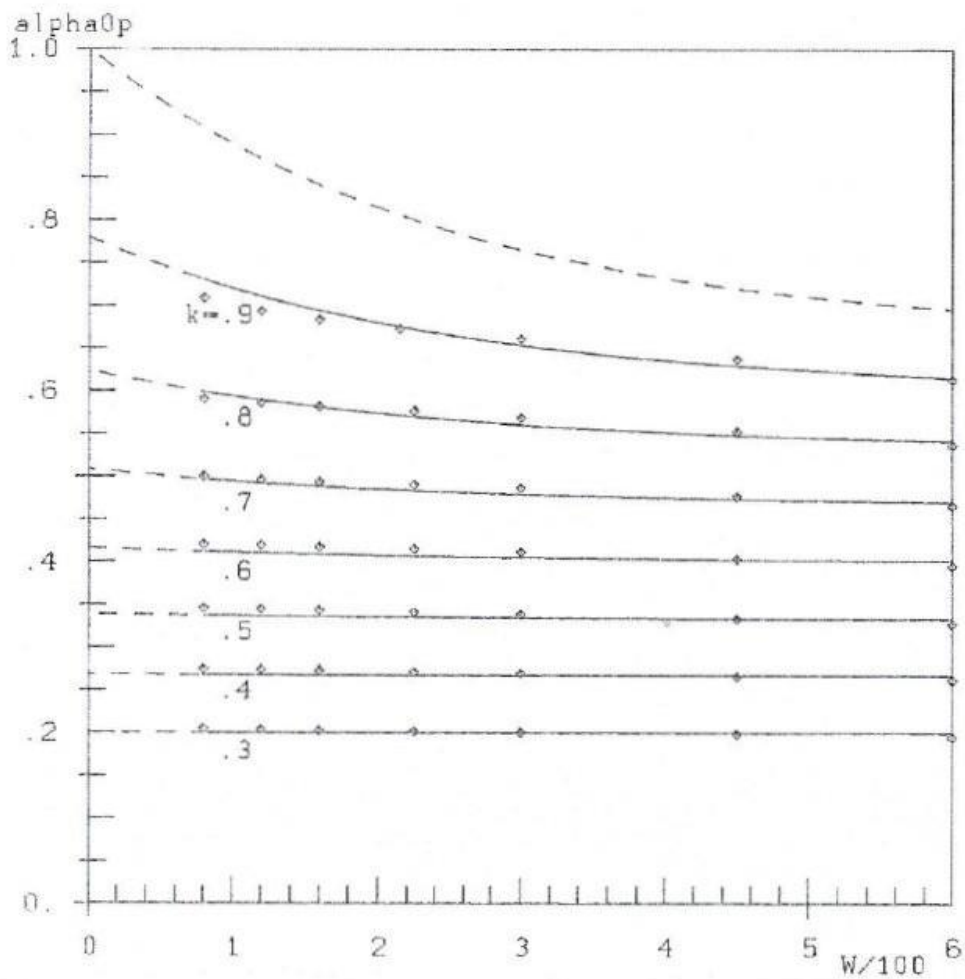


Figure 5b: experimental points $(W/100, \alpha_0^P)$ and hypothetical curves $(W/100, \hat{\alpha}_0^P)$

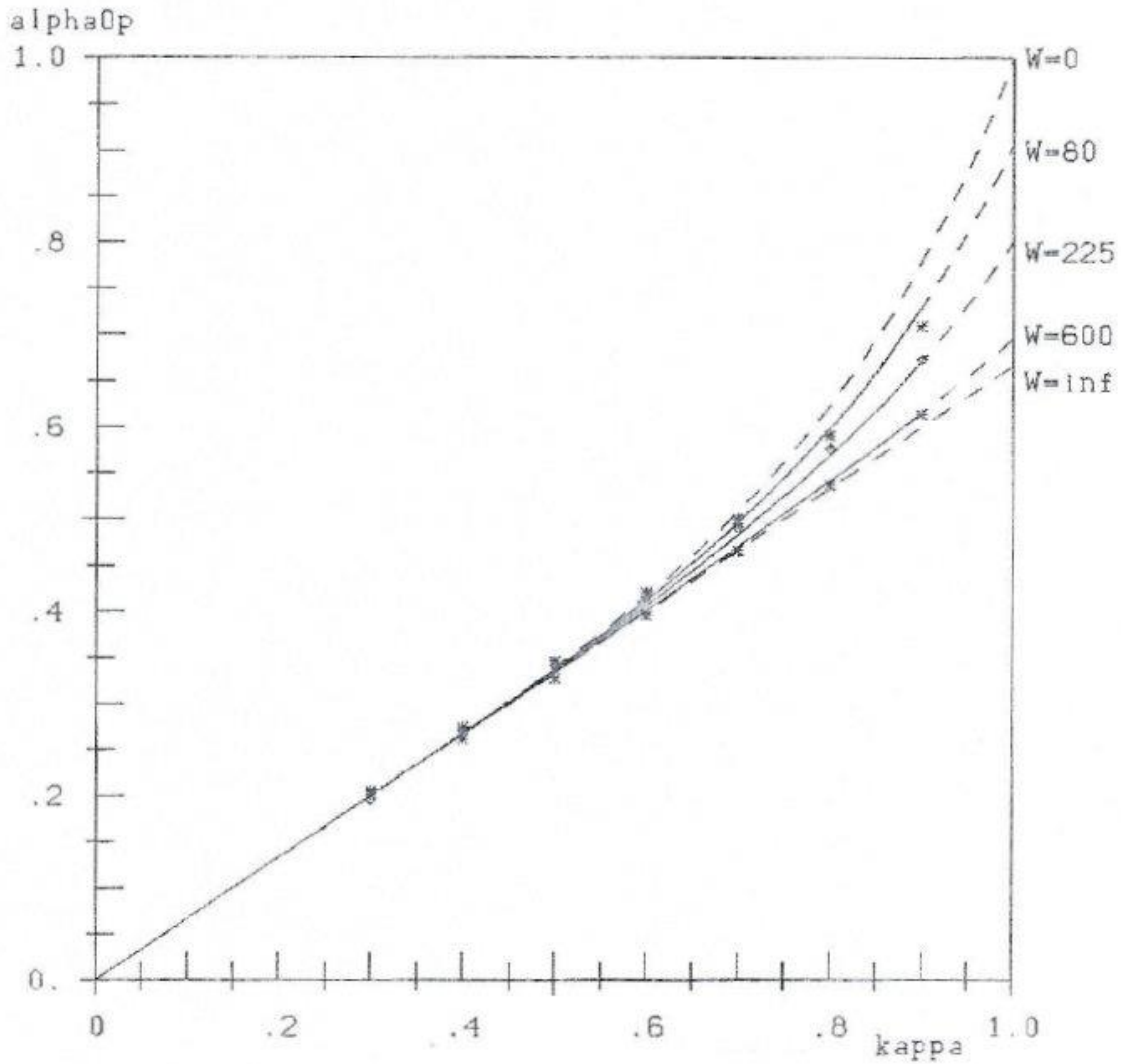


Fig. 5c: experimental points (κ, α_0^p) and hypothetical curves $(\kappa, \hat{\alpha}_0^p)$

Hypothesis H3 for the stress-strain relation with **non-centric** loading:

$$\hat{\kappa} = a \varepsilon_0 e^{-b \varepsilon_0}$$

„The stress-strain relationship for non-centric loading is the same as that for centric loading“.

See [4].

The (now 63) experimental data (ε_0, κ) are given in table 3a, lines 1 and 2, the hypothetical results $\hat{\kappa}$ in line 3. The parameters a and b for 7 values of W are given in table 3b. Figure 6b shows the correspondence of data and hypothesis.

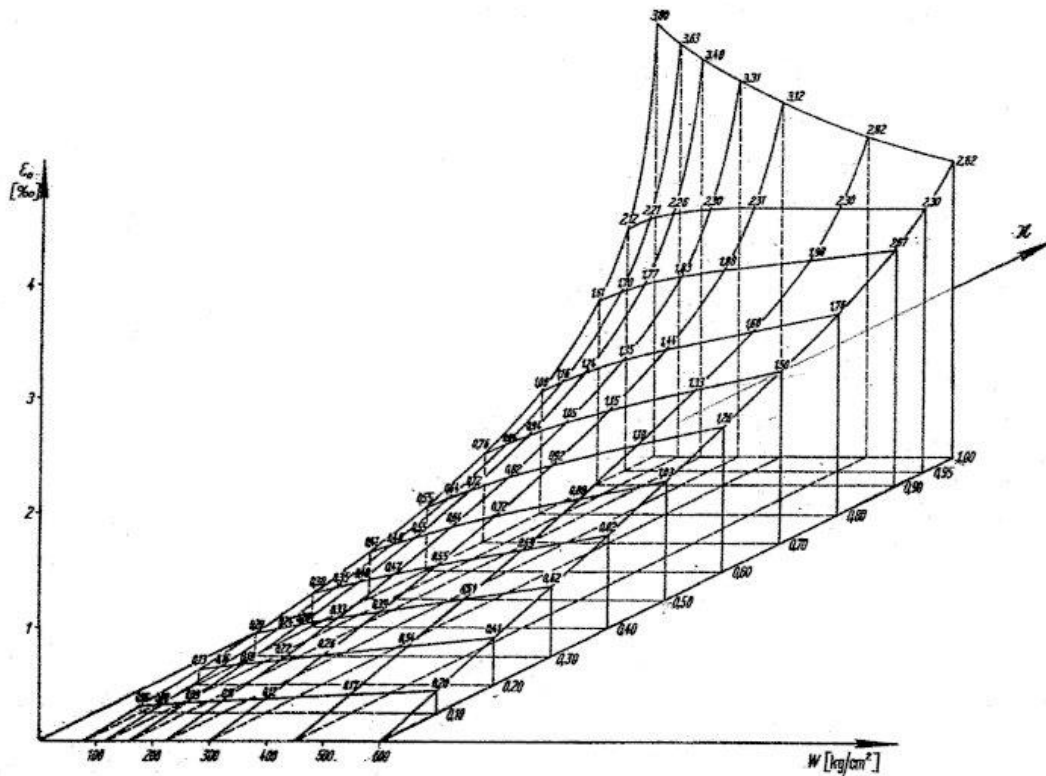


Fig. 6a: Dependence of the edge compression ϵ_0 on the strength of the cube and the degree of loading κ of the bending pressure body

Table 3a: first lines: experimental values ϵ_0 , second lines: experimental values κ , third lines: hypothetical values $\hat{\kappa}$

W=80	0.06	0.13	0.20	0.30	0.41	0.55	0.76	1.06	1.62
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.0903	.1869	.2747	.3859	.4906	.6005	.7230	.8282	.8770
W=120	0.08	0.16	0.24	0.35	0.48	0.64	0.86	1.16	1.70
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.1029	.1968	.2823	.3872	.4939	.6023	.7158	.8168	.8856
W=160	0.09	0.18	0.28	0.40	0.55	0.72	0.94	1.24	1.77
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.1021	.1956	.2900	.3909	.5000	.6028	.7076	.8074	.8920
W=225	0.11	0.22	0.33	0.47	0.64	0.82	1.05	1.35	1.83
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.1066	.2041	.2931	.3950	.5030	.6002	.7018	.8014	.8987
W=300	0.12	0.26	0.39	0.55	0.72	0.92	1.15	1.44	1.88
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.1005	.2084	.3001	.4024	.4993	.5990	.6964	.7958	.9044
W=450	0.17	0.34	0.51	0.69	0.88	1.10	1.33	1.60	1.98
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.1093	.2119	.3080	.4031	.4964	.5959	.6906	.7905	.9121
W=600	0.20	0.41	0.62	0.82	1.03	1.26	1.50	1.76	2.07
	.1	.2	.3	.4	.5	.6	.7	.8	.9
	.1036	.2085	.3095	.4022	.4959	.5945	.6930	.7947	.9095

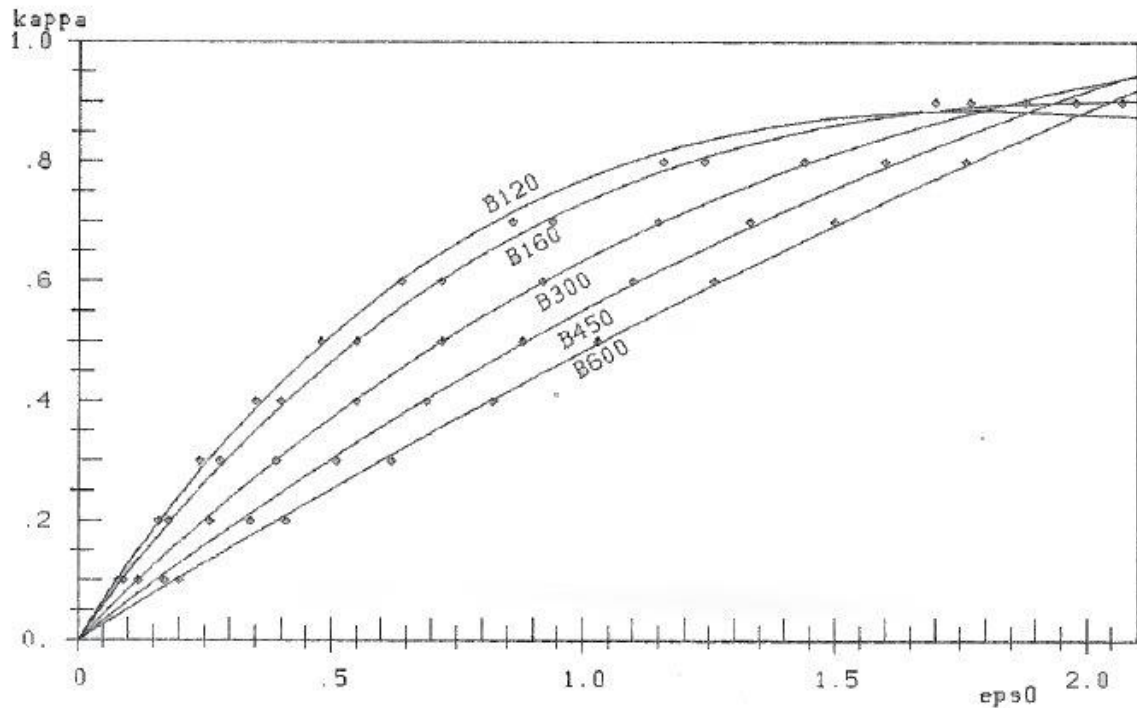


Fig. 6b: experimental points (ϵ, κ) and hypothetical curves ($\epsilon_0, \hat{\kappa}$)

Table 3b: Parameters of hypothesis $\hat{\kappa}$

	a	b
W=80	1.566	.6559
120	1.345	.5579
160	1.186	.4835
225	1.012	.3951
300	.8702	.3153
450	.6634	.1842
600	.5271	.08793

In a following part III I will bring a theory of the distribution of the stress across the bending pressure zone – on the basis of these hypotheses.

Acknowledgement

I state, that this analysis could not have been done without the grand work of Rüsçh and his assistants. In hard times – before 1955 . He initiated, carried through and analysed – with the then available means – an immense scientific program, the results of which are concentrated in his 3 resulting figures 112/113/114, the basis of this work.

References

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[4] Schneeberger H. (2020). [Hooke's Law: Nonlinear Generalization and Applications. Part 2: Parameters of the Bending Pressure Zone of a Beam](#)